Experimental Investigation of performance and emission characteristics of Ethanol blended diesel with zinc oxide nanoparticles.

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Abstract— Diesel engines are widely used in the field of transportation and power generation because of its better fuel economy. Due to dwindling of petroleum resources suitable alternative fuel has to be replaced in the place of existing petroleum products. Global demand in thrusting researchers to focus on alternate fuel which serves better than diesel. The main work of our manuscript deals with the comparison on effect of zinc oxide nanoparticle with ethanol blended diesel in two suitable proportions. Firstly 200ppm of zinc oxide nanoparticle blended with 15% by volume of Ethanol with Diesel (D85 E15 Z 5). Another blend with 5% of Ethanol blended with Diesel (D95 E5 Z2) along with 500ppm of zinc oxide nanoparticle. The blend were tested in different load conditions in kirloskar single cylinder water cooled naturally aspirated compression ignition engine. The blend were compared with base diesel fuel which lead to a conclusion where there was a reduction in CO_{2} ,CO,HC and there was a considerable increase in the NO_x, and Brake Thermal Efficiency(B.T.E).

Keyword- Zinc oxide nanoparticle, Ethanol blended diesel, Diesel engine, performance and emission.

I. INTRODUCTION

India has GDPA growth of 10% for the last year which in turn demands to growing industrial needs. So transporting raw materials and finished products through the country plays a major role. The field of power production also concerned with flourishing industrial growth as per stated by honorable Prime Minister of India Mr.Nardendra Modi affirming the fact that India will become 3rd largest economy of the world at 2030. And it will become the top 3 preferable manufacturing destination by corporate company by the end of 2020. So this directly reflects the demand for the transportation and power generation in the preceding years.

These two areas are precisely dominated by diesel engine because of their high thermal efficiency but this growing demand also reflects in the need of importing petroleum products from the other countries. So this cause triggers researchers to find a suitable alternative which serves and a better performing fuel which helps in reducing the emission and increase in the fuel efficiency. Ethanol serves as suitable alternative because it can be easily produce carbon foot print blending along with diesel will lead to a considerable amount of reduction in emission.

The practice of using nanoparticles over the years has increased the knowledge on how the nanoparticle influence the emission parameters of diesel engines. An extensive literature survey leads to a conclusion that zinc oxide nanoparticle reduces Carbon Mono Oxide (C.O.) and Hydro Carbon (H.C.) by considerable amount there by increasing the Brake Thermal Efficiency(B.T.E.) by small fraction. The objective of this experimental study is to investigate the characteristics of exhaust emission and engine's performance for two ethanol diesel blends.

II. SYNTHESIS OF BLENDED FUEL:

Ethanol was used to reduce the amount of carbon footprint in the fuel but in normal conditions blending of ethanol and diesel won't be completely miscible because of the surface tension between the two fuels. Inorder to make the fuel completely miscible surfactant: span 80(3ml) was used for one liter of each blended fuel. A magnetic stirer which emphasis high-speed mixing was used to blend the ethanol and diesel.

Four blend proportions were primarily prepared, then suitable zinc oxide nano particle were added. The overall two dosages were used :one with 200ppm and other with 500ppm. The required doing level were measured by the help of precession electronic weighing scale and then the fuel were mixed by means of ultrasonic agitator. The blended fuel were given at a constant agitation of 45minutes to produce an uniform blend and then the fuel were left for phase separation process.

S.NO	Diesel Proportion(ml)	Ethanol Proportion(ml)	ZnO2 Nano Particle (gm)
1	950	50	0.2
2	950	50	0.3
3	850	150	0.3
4	850	150	0.5

TABLE I. Blend Proportions

It was found that blend with 0.2 and 0.5 Zinc oxide nanoparticle had a stable time period of more than one week. Those two blends were used to calculate the performance and emission characteristics.

III.STABILITY DETERMINATION:

The Two blends which had 0.2 and 0.5 gm was found to be the most stable fuel, these Two Blends showed meager phase separation of ethanol and Diesel after one week. In order to find the dispersion of Nano particles in the Blends SEM(Scanning Electron Microscope) test was carried out, Both the blends showed different type of Nano particle dispersion.

Parameters	Details	
ACC Voltage	0.3 to 30 KV	
Magnification	5X to 300,000 X	
Resolution	3.0nm (30Kv HV mode)	
	10nm (3Kv LVmode)	
	4.0nm (30 Kv LVmode)	
Standard detectors	SE ,BSE	
Specimen stage	Fully motorized, 100/50 XY movements	
PC	Latest PIV	
Coating Unit	Ion sputter coater with Gold target	
Chamber viewing	IRCCD camera	
EDS detector system	LN2 free, Peltier cooled, 139 eV	

TABLE II. Scanning Electron Microscope specification.



Fig. 1,2: Blend with 0.2 gm ZnO₂ in SEM test under 10um and 5um

The first blend showed moreover a even dispersion of nano particle under 10um, but there are some lumps of Nanoparticles which are still not completely miscible with the fuel. These lumps will results in settling back in the combustion chamber which would result in acting as a heat sink which will be helpful in higher operating condition but in prolonged usage it will result in engine starting problem, due to the loss in Transcent Temperature.



Fig. 3, 4, 5: Blend with 0.5gm ZnO₂ in SEM test under 20um and 2um.

The Second Blend which has more quantity of Zinc oxide particle showed lot of coagulation and uneven dispersion of the particles. This fuel blends will result in uneven emission graphs, due to the irregularity found through out the fuel. This fuel would also result in cloging of injectors and it can be prevented by changing the Diesel filter regularly. Since the Nano particles are not spread evenly they will not help in reducing the operating temperature hence NO_x parameter will be high for these fuels.

Experimental Setup:



Fig. 3 : Diagram of Experimental Apparatus.

TABL	E III.	Engine	Parts
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NO	Description	
1.	VCR Engine	
2.	Pressure Sensor	
3.	Crank angle encoder	
4.	Speed Sensor	
5.	Air flow sensor	
6.	Exhaust gas line	
7.	Gas Analyzer	
8.	Gas Calorimeter	
9.	Load Sensor	
10.	Dynamometer	



Fig. 6: Kirloskar DM 10 Engine

Our experiment is conducted in Hindustan Institute of Technology and Science, (Chennai) Single cylinder water cooled naturally aspirated compression ignition engine was used to perform the test where the engine has maximum Horse power(H.P.) of 10h.p. and the output shaft is connected to electrical dynamometer which has maximum capacity of 24 amps. The engine uses mechanical injector and it is a constant engine which is designed to run at 1500 rpm.

Make & Model	Kirloskar DM 10	
Bore/Stroke	102*116mm	
Speed	1500	
B.H.P	10hp	
Swept Volume	661cc	
Compression Ratio Range	17.5:1	
Fuel	Diesel	
Torque	0.048 KN – m	
Dynamometer	Electrical	

TABLE IV. Engine Specification.

Atlon 5 gas analyzer was used to measure the emission parameter of the exhaust gas. This gas analysis measures 5 parameters (Carbon Mono-oxide(CO), Carbon Dioxide(CO₂), Hydro-Carbon(HC), Oxides of Nitrogen(NO_x) and Oxygen(O₂).

Make	AVI	
Model	DIGAS 444	
Display	LCD	
Interface	RS 232	
Power Consumption	25 Watts	
Operating Voltage	230Volt 55Hz	

TABLE V. Gas Analyzer Specification.

In order to analyze the performance and emission characteristics the (D85 E15 Z2), (D95 E5 Z5) were introduced to engine in all quarter, half, three fourth and full load conditions. And the required parameters are recorded.



Fig. 7: Atlon Gas analyzer

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IV.RESULTS AND DISCUSSION:

ENGINE PERFORMANCE:

When the two blends (D95 E5 Z2), (D85 E15 Z5) were tested, the latter blend which has more nanoparticle produced slightly higher Brake Thermal Efficiency which was around 2% this is because of the presence of higher oxygen content in the blend. This actually influences better combution characteristics. The first blend which has a only 5% of ethanol, the efficiency drops in part load and in full load conditions the efficiency eclipses the base fuel reading. There was a marginal increase of 1% BHP in full load condition.



Fig. 8: Comparison of Base fuel with D95E5Z2 and D85E15Z5 for Break Thermal Efficiency and load

Engine performance directly influences the total fuel consumption of each blend as the latter blend which has more ethanol content consumes less fuel when compared with base fuel because of higher combustion efficiency. The overall Total fuel consumption of both blends in part and full load was found to be 2 to 3 % less compared with base diesel reading. But at low loads the Total Fuel Consumption of the blended fuels were high due to the low calorific value of the fuel.



Fig. 9: Comparison of Base fuel with D95E5Z2 and D85E15Z5 for Total Fuel Consumption and Load.

EMISSION CHARACTERISTICS :

Carbon di-oxide emission of the blended fuel found to be 10 to 15 % higher than the base fuel reading at full load conditions. At low loads and part load condition there was a marginal increase of within 5 % this is because of effect of nanoparticle which interact with combustion at low loads the influence of nanoparticle is high. At high loads since more quantity of fuel is injected more CO_2 is produced.



Fig. 10: Comparison of Base fuel with D95E5Z2 and D85E15Z5 for Carbondi-oxide and load

Oxide of Nitrogen emission for the blended fuel found to be to 9 % higher than the base fuel reading at full load conditions. At low loads and part load condition there was a marginal increase of within 15 % this is because of effect of better combustion characteristics which leads to higher operating temperatures which is influenced by the presence of additional oxygen in the blend Nanoparticle which acts as a heat sink reduces the formation of NO_x . But still it goes on to be higher than the base fuel reading.



Fig. 11: Comparison of Base fuel with D95E5Z2 and D85E15Z5 for Oxides fo Nitrogen and Load.

Hydro-carbon which is primarily formed because of incomplete combustion, since the combustion has been improved in the general fuel the amount of (HC) formed was found to be less in all load conditions in the blended fuel. The average reduction of HC is found to be 30% and 15 % respectively.



Fig. 12: Comparison of Base fuel with D95E5Z2 and D85E15Z5 for Hydro-carbon and load.

CO formation is primarily because of unavailability of oxygen the blended fuels which has better oxygen content helps influencing in the complete combustion which results in overall decrease in the CO emission by upto 11% in all load conditions



Fig. 13: Comparison of Base fuel with D95E5Z2 and D85E15Z5 for Carbon mono-oxide and Load.

V. CONCLUSION:

The performance and emission characteristics for the blended fuel (D95 E5 Z2), (D85 E15 Z5) which was tested in a single cylinder compression ignition engine are summarized as follows :

- 2% increase in Break Thermal Efficiency was found in part load conditions and 1% increase in full load condition for the blended fuels.
- In a comparative note with base diesel reading there was around 2 to 3 % decrease in the Total fuel consumption in part and full load condition of the blended fuels.
- While the CO₂ footprint in the blended fuel were 10 to 15 % increase when compared to the base fuel reading at full load condition.
- Averagely the reduction of HC in the blended fuel was found to be 30 to 15 % for the fuel blends.
- Thus a overall decrease in the CO emissions was 11% in all load conditions for the blends.

VI. REFERNCE:

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